



IN THE UNITED STATES PATENT AND TRADEMARK OFFICE

Applicant(s): Heiland et al.
Serial No.: 10/810,361
For: FAST SWAPPING STATION FOR WAFER TRANSPORT
Filed: March 26, 2004
Examiner: Not Yet Assigned
Art Unit: 3652
Confirmation No.: 1260
Customer No.: 27,623

Attorney Docket No.: 2133.031USU

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We are enclosing a certified copy of the priority document, German Application No. 103 14 383.1 filed 28 March 2003, for filing in the above noted application.

It is respectfully requested that this application be passed to allowance.

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Date: July 27, 2004

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Prioritätsbescheinigung über die Einreichung einer Patentanmeldung

CERTIFIED COPY OF PRIORITY DOCUMENT

Aktenzeichen: 103 14 383.1

Anmeldetag: 28. März 2003

Anmelder/Inhaber: Integrated Dynamics Engineering GmbH,
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Bezeichnung: Fast swapping station for wafer transport

IPC: H 01 L, B 25 J, B 65 G

Die angehefteten Stücke sind eine richtige und genaue Wiedergabe der ursprünglichen Unterlagen dieser Patentanmeldung.

München, den 2. April 2004
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Stremme

Integrated Dynamics . . .

FAST SWAPPING STATION FOR WAFER TRANSPORT.

BACKGROUND OF THE INVENTION.

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1. FIELD OF THE INVENTION.

The invention relates to an apparatus for swapping at least one disklike member according to claim 1, and a 10 method for transporting wafers as claimed in claim 14, and a handling line for carrying out the method of claim 15 as defined in claim 16.

2. DESCRIPTION OF RELATED ART.

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Apparatus for accepting and transferring wafers are well known in the art. For instance, robot handlers are commonly used to move materials, e.g. semiconductor wafers, between different stages of a wafer fabrication process. In 20 this regard, robot handlers might be used to move the wafer from a plasma etch station in a cluster tool to a deposition station or from a manufacturing station to a testing station or metrology tool, wherein the wafer is positioned onto a chuck. In this kind of handling system the throughput depends 25 strongly on the time the metrology tool has to wait for material handed over by the robot handler.

A typical transporting or handling scenario for a wafer handling system is the following:

- a) the handling system, typically a one-arm-robot-handler gets a new wafer,
- b) the handling system moves and loads the wafer onto the transfer position for transferring the wafer for example into the metrology tool,
- c) the wafer gets moved into the tool for processing or measuring; after this the wafer gets back to the transfer position or unload position which is typically the same as the load position,
- d) the handling system, i.e. the robot handler, picks the wafer and unloads the wafer to the next position,

The cycle starts from step a)

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During the processing of the wafer in the metrology tool the handling system or robot waits or is doing different tasks. But for doing different tasks the handling system has to do motions without wafer. This however is ineffective and forms a bottleneck of the transport system which limits the throughput.

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The problem might be solved by employing a so called dual-arm or paddle-robot. Since the dual arm robot can handle two wafers at the same time. However, such a handling system has a increased footprint, since the arms increase the sweep radius of the robot, which is caused by the space that is needed horizontally to swap the wafer at the desired station.

30

Yet, space in a clean room environment, which is commonly needed for wafer production, is a scarce resource. Therefore, an increase in food print of the handling system would raise production costs considerably. Moreover these kind of robots are more complex in handling and their purchase price is high.

35

Therefore a solution is needed which on the one side increases the throughput and on the other minimizes the space consumption and the additional footprint, respectively,
5 needed.

These and other disadvantages have lead to the object of the present invention to provide an apparatus as a part of a handling system which avoids to use complex handling
10 systems with a big footprint but nevertheless can increases the throughput considerably.

SUMMARY OF THE INVENTION.

15 The inventive solution is obtained by the provision of an apparatus according to claim 1

Advantageous further developments are part of the
respective dependent claims.

20 Thus far, in general, the invention is an apparatus for swapping at least one disklike member, e.g. a wafer, the apparatus comprises at least two tongslike arms for accepting and holding the disklike member and a driving-mechanism adapted to drive said arms, wherein the mechanism is adapted
25 to provide a first and a second movement of the arms, whereby the first movement comprises a vertically oriented movement of the arms from an up to a down position and vice versa, and whereby the second movement comprises a horizontally oriented
30 tongslike movement of said arms.

The inventive apparatus, for example, can be advantageously simply arranged in-line or in series to an one arm handling system and a load position of, for instance, a
35 chuck of a motion system which transfers a disklike member or

a wafer, e.g. into a testing station. In this regard the inventive apparatus is able to accept, hold or store the disklike member with its tongslike arms, being driven by the driving mechanism, from the handling system, whereby, after 5 having placed the disklike member or wafer into the arms of the inventive apparatus, the unloaded handling system can pick another wafer from, for instance the chuck, coming out of the testing station. If these steps are completed, the inventive apparatus is highly advantageous able to transfer 10 or place or swap the stored wafer with its tongslike arms on the load position or chuck to be transferred into the testing station. Of course the just described cycle works also the other way round. Thus, any motion of the handling system without wafer can be avoided if the inventive apparatus is 15 applied.

It should be emphasis that according to the invention the term disklike member, in its meaning, comprises any kind of member which can be swapped or transferred by means of the 20 inventive apparatus, or which is adapted to be used within the inventive process. Thus, in the inventive sense disklike members can be, for instance, round like a wafer, or rectangular like a plate, or can be of any appropriate shape.

25 With regard to an advantageous further development of the inventive apparatus a lever apparatus and/or spindle means is provided which controls or governs the movement of the tongslike arms. Thus, the lever apparatus and/or spindle means together with the inventive driving mechanism, at least 30 guarantees a movement of the tongslike arms into one hold and into one release position and vice versa. The hold position is characterized by the fact that in this position the arms are ready to accept and/or to hold a disklike member or wafer, whereas in the release position the arms are opened 35 such that a disklike member can be charged or discharge by

the arms. Thereby, it is guaranteed that especially in the case when the arms charge or grip a disklike member or wafer the distance between the arms and the arms and the dislike member, respectively, is such that there will be no damage of 5 the disklike member.

According to an additional advantageous further development of the inventive apparatus the driving mechanism is separated into an elevation contrivance and a manipulator 10 drive. The manipulator drive causes the arms to perform a tongslike move in a plan, whereas the elevation contrivance preferably moves the arms and/or the manipulator up and down, i.e. in a direction preferably perpendicular to the plan defined by the tongslike movement of the arms. 15 Advantageously, both movements, and thus both apparatus can be controlled independently.

Yet, with regard to another preferred embodiment of the invention, it is provided that the driving mechanism or 20 the manipulator drive comprises the functionality to move the arms into two different hold positions and into two release positions corresponding to the hold positions. This kind of functionality gives the possibility to use the inventive apparatus for different kind of disklike members, e.g. for 25 wafer with different diameters. So far, the inventive apparatus might be for example used for wafer seizes with a diameter of 200 mm or of 300 mm.

In connection with this, the invention also provides 30 extension members that are part of the arms, and which advantageously allow to flexible adjust the feed opening or the distance between the arms. This opens up the possibility to use the inventive apparatus for swapping disks with not only two or three different diameters but with all kind of

diameters.

Another structural element of the invention which positively further develops the inventive apparatus is given if the apparatus comprises gripping means which helps the arms to grip the disklike member. The gripping means contacts the edges of the disklike member, when it is held or gripped by the inventive apparatus or tongslike arms. For that purpose advantageously the gripping means comprises at least one grooved circular ring section which is adapted to the radius or diameter of the wafer to be gripped.

For to provide a compact assembly the inventive apparatus comprises a housing, wherein at least a part of the driving mechanism for the tongslike arms is accommodated. Thereby openings are provided through which the arms extend for to grip the disklike member. Advantageously, moreover, it is given that the arms are formed in a way that enables the apparatus and the arms respectively to be front- or to be back-loaded. Front-side in this respect is the side to which the arms extend to, and back side is the side in opposition to the front-side.

According to another further development of invention the inventive apparatus or driving mechanism comprises at least one driving motor, which one the one side drives the vertical movement of the tongslike arms via the lever apparatus and/or spindle and which on the other side also drives the horizontal movement of the arms.

30

Highly advantageously, according to the invention, the inventive apparatus also comprises sensor means for detecting the disklike member, i.e. for to tell the apparatus whether there is a disklike member or not inside the inventive apparatus and/or for detecting the position of the

5 tongslike arms. For the latter purpose sensors, for instance, can be arranged in pairs with respect to the arms such that they are able to indicate at least two different relative positions of the arms to each other. In this respect, one of the sensor, for example, could indicate the hold position of the respective arm and the other the release position of the respective arm. The same applies if the inventive apparatus is dimensioned for disklike members of different diameter, whereby for each disklike member a hold and a release 10 position can be defined via the arrangement of sensors or via sensor readings. Moreover, sensors could be provided which indicate the height of the arms with respect to a defined level.

15 According another aspect of the invention the inventive apparatus favorably comprises or is connectable to control means for controlling the movement of the arms and/or motor drives etc. For that purpose appropriated interfaces and at least one micro-controller are/is provided.

20 Additionally to the above stated, it is a further object of the present invention to provide a method which by use of the inventive apparatus avoids to use complex handling systems but nevertheless increases the throughput 25 considerably.

The solution for this problem is given by the method as claimed in claim 15.

30 Therein a method for handling or transporting disklike members, e.g. wafers, is defined, wherein an apparatus as described above is positioned into an exchange-region for exchanging dislike-members, for that purpose a first disklike member with first transport means from a first 35 position is transported to the exchange region, at the

exchange region the first disklike member is loaded into the inventive apparatus, after or parallel to the latter load or even before the latter load a second disklike member with second transport means is transported to the exchange region, 5 the second disklike member, after having unloaded the first transport means, the second disklike member is loaded from the second transport means to the first transport means, and after having unloade the second transport means, the first disklike member gripped, loaded or charged by the inventive 10 apparatus is transferred from the inventive apparatus to the second transport means, if this is done the second disklike member can be transported to a second position by means of the second transport means. The cycle can start again.

15 An even further object of the invention is to provide a handling line to accomplish the just described method. The inventive handling line comprises respective means for carrying out the method. Thus, in particular, it might comprises a x-y-stage or a chuck and/or a robot to replace 20 either the first or the second transport means.

BRIEF DESCRIPTION OF THE DRAWING.

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The invention together with additional features and additional advantageous thereof will be best understood from the following description.

30 It is shown:

Fig. 1 a cross-sectional view of one embodiment of an apparatus according to the invention wherein the tongslike arms are in the up position,

35

Fig. 2 the cross-section as shown in Fig. 1, wherein the tongslike arms are in the down position,

5 Fig. 3 a bottom view Fig. 1 including the wafer the arms and the grippers of the arms,

Fig. 4 a cam disk top view with two cam rings,

10 Fig. 5 a topography or profile of the cam rings of the cam disk according to Fig. 6

Fig. 6 a place cycle of the arms,

15 Fig. 7 a moving flow chart of the arms,

Fig. 8 a perspective view of the front side of another embodiment according to the invention,

20 Fig. 9 a perspective view of the back side of the embodiment of Fig. 8,

25 Fig. 10 a schematic top view of the manipulator drive together with the tongslike arms of Fig. 8 except that the manipulator drive just comprises one actuator for driving the lead screws and the tongslike arms respectively,

30 Fig. 11 a schematic top view as shown in Fig. 10 except that the manipulator drive comprises a manipulator-belt-drive assembly for driving the lead screws and the tongslike

arms respectively,

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Fig. 12 a schematic top view of another embodiment of the manipulator drive and the tongslike arms of an inventive swapping apparatus, wherein the manipulator drive comprises a gear drive and lever apparatus for driving the tongslike arms.

10

Fig. 13 the handling process if an apparatus according to the invention is used, in case the inventive apparatus is first front-loaded,

15

Fig. 14 the handling process of Fig. 8 with the difference that the inventive apparatus is first back-loaded.

DETAILED DESCRIPTION OF THE DRAWINGS.

Figures 1 and 2 are showing a design of the inventive apparatus generally referred to hereinafter to be a Fast swapping station (FSS-1). The inventive fast swapping station comprises a driving mechanism 100 and tongslike arms 107a and 107b. The propulsive power of driving mechanism 100 is provided by motor 101. The motor 101, which is fixed to the top part of the housing 102 is mounted in the center of the housing 102 of the driving mechanism 100 and rotates the cam plate 104. The housing 102 has a cylinder box structure with openings 102a and 102b in its cylinder-jacket, whereby the openings 102a and 102b are arranged opposed to each other. The driving axis 101a of the motor 101 is fixed in the mounting hole 104c of the plate 104. The disk or plate 104 has two higher circles or rings 104a and 104b with a changing topography in height which reflects the cam structures 103a and 103b on the disk 104 (see also Fig. 4). The rotating plate 104 and the driving axis 101a are supported by the bearing 105 at the center of the lower part or bottom side of the housing 102. In side the housing, centrally arranged, a space 112 is provided which accommodates the motor 101. The space is formed by a wall 113 surrounding the motor and is adapted to the geometry of the motor, e.g. cylindrical, together with the upper part or top surface of the housing 102, to which the wall 113 is integrally attached to, and from which it extends down to the bottom of the housing and the plate 104. The wall 113, being in touch with the motor 101 at its inner side, is the inner part of a sliding bearing 106 at its outside. The counter or outer part of this sliding bearing 106 is the inside of an opening of the support structure 110 (110a and 110b) which is riding with two rollers 117a and 117b on the inner circle 104b with the cam structure or contour profile 103b. This support structure 110 holds two arms 107a and 107b. The arms 107a and 107b are

rectangular in shape and comprise two legs 115a, 115b and 116a, 116b each, namely upper legs 115a and 116a and side legs 115b and 116b.

5 For the sake of simplicity the structure and function of only one of the arms 107a and 107b is described further, whereby the arms 107a and 107b are in structure and functionality the same except that they are arranged in opposition or mirror relationship to each other.

10

The upper leg 115a of the arm 107a has been inserted into or extends through the opening 102a of the housing 102, and is moveably attached to the fulcrum 114a of holding arm 110a of the support structure 110 to form a lever apparatus, which allows the upper leg 115a under defined circumstance, which will be described hereinafter, to change its angle with respect to the horizontal line and the holding arm 110a respectively. The upper leg 115a with its end inside of the FSS-1 housing 102 has a roller 108a rotatably fixed at its end, whereby the roller 108a is supported by the second topography 103a of the rotating plate 104. The upper leg 115a, and thus the support structure arm 110a too, is held down or pushed against to the contour circles 104a or cam structure 103a with the spring 109a. For this purpose the spring 109a is fixed to the upper leg 116a and the bottom part of the housing 102 at a point between the roller 108a and the bearing fulcrum 114a depending on the force to be exerted on to the upper leg 115a and the contour profile 103a, respectively.

20

25 Thus, by rotating the plate 104 the rollers 117a and 117b of the support structure 110 will follow the topography of the cam structure 103b, and the rollers 108a and 108b of the arms 107a and 107b will follow the topography of the cam structure 103a, whereby the arms 107a, 107b perform a

30

tongslike movement. It is the difference between the two profiles of the two cam structures 103a and 103b that causes a change in the relative position between the support structure 110 and the arms 107 (107a and 107b). In this regard Figure 1 shows the support structure 110 to be pushed up upon rotation of plate 104, and the arms with their upper legs 115a and 116a are pushed up to the same height as well. This is the situation in which the arms can hold a wafer 118 or, in case there is no wafer, can accept or store a wafer 10 from a robot handler of a wafer handling system.

Figure 2 shows the support structure down upon rotation of the plate 104, and the arms support 103a is even more down, because at the angular position of plate 104, shown in Figure 2, the rollers 108a and 108b of the arms 107a and 107b, respectively, are pushed down to a lower height than that of rollers 117a and 117b which results in a counter clockwise movement of the arm 107b and a clockwise movement of the arm 107a, up to a defined angle, that depends on the cam structure differences in height at the down position, whereby the side legs 115b and 116b are pushed aside like the arms of a tongs. In this case the arms 107 (107a and 107b) are opened in a down position to release the wafer 118 to a chuck 120 or to accept the wafer 118 from the chuck 120.

25

The grippers or retaining zones 119 (119a and 119b) of the arms for to hold the wafer consists of two circular ring or rim sections 119a and 119b, which radius is adapted to that of the wafer (Fig. 3). In the cross-sectional view of Figures 1 or 2 it can be seen that the ring sections 119a, 119b comprise retaining grooves, whereby the cross-section of the grooves deviates slightly from a "L". In this respect, the vertical part makes an angle with the lower part which is bigger than 90°. Thus, the wafer can be easily gripped into 35 the retaining zone 119 without pinching and damaging it. In

the hold position of the arms 107 the wafer sits on the lower part of the groove and the vertical part or raising edge of the groove keeps the wafer 118 at a defined position, i.e. radius, during motion. This can be even better seen in Fig.

5 3. Therein the dashed lines mark the end of the grippers or retaining zones .

10 The gripping mechanism of the FSS-1 is the like that the arms 107 open wide enough and closes slowly enough that when the arms 107a and 107b are pushed upwards to the up or hold position, whereby the grippers 119 pass the wafer without touching it on its way up.

15 In order to achieve this motion a profile or cam structure 103a and 103b of the contour is provided as shown in Figure 5. Fig 5 displays in part a projection of the cam 103a and 103b of the profile rings 104a and 104b (Fig. 4). From 0° to 10° both contours 103a, 103b are stable at a height of 10mm. From 10° to 80° both profiles drop parallel, 20 whereby in the projection of Fig. 5 the contour lines coincide. In this way the rotation of the plate or disk 104 leads to a down movement of the arms without opening of the arms (see also Figures 6a, 7a and 6b, 7b). After 80° the outer contour 103a drops more than that inner contour 103b, 25 i.e. down to minus 2mm. Thus, the arms 107a and 107b turn around the fulcrums 114a and 114b to open the grippers 119 and the arms, respectively, (Figure 6c, 7c). From 100° to 170° both profiles moving parallel with a constant difference so that the arms go up, but stay open (Figure 6d, 7d). From 30 170° to 180° the outer profile 103a comes to the same height as the inner one 103b. The arms close again (Figure 6e, 7e).

35 A further movement of the plate 104 in the same direction up to a 360° would repeat the transfer motion described above. However, if the motion of disk 104 is

reversed instead of the described place cycle a pick cycle would be initiated starting with the end position of the place cycle and ending with the start or up position of the place cycle.

5

This functionality is also demonstrated in Figures 7a to 7e the arms 107 (107a and 107b) first move down from a height of 10mm to the zero level point, which marks the maximum down positioning of the arms 107. A further rotation of the disk leads to the opening of arms and to the placing of the wafer on the chuck. By rotating backwards, i.e. from the 360° position to the 180° position or from the 180° position to the 0° position a wafer can be picked from the chuck and can be brought to the upper or storage position.

15

Thus the functionality allows a fast swapping of wafers at the chuck with a handling tool that can handle only one wafer: Thereby the robot with only one arm puts the new wafer in the FSS-1, which is closed, by moving the wafer into the correct center position with an increased height to overcome the outer rims of the wafer retainer structure 119a and 119b. Then the robot (not shown) places the wafer down onto the support structure. Now the robot picks the old wafer from a chuck. After the wafer is removed from the chuck the FSS-1 places the wafer onto the chuck 120 by moving the sequence from Figure 6a to 6e or 7a to 7e.

The FSS-1 is now ready to accept the next wafer and the fast swapping cycle can start from the beginning.

30

In the following reference is made to Fig. 8, wherein a front side perspective view of a further embodiment of the inventive apparatus FSS-2 is shown. FSS-2 displayed in Fig. 8 comprises tongslike arms 207 (right arm 207a and left arm 207b) and a driving mechanism 201. Each of the arms 207

consist of a guiding member 208 (208a right arm and 208b left arm), an extension member 209 (209a right arm and 209b left arm), a leg 210 (210a right arm and 210b left arm), and a gripper 99 (99a right arm and 99b left arm). The driving mechanism 201 is made up of an elevation contrivance 202 and a manipulator drive 203. It is the purpose of the manipulator drive to appropriately control, govern or drive the arms 207a, 207b, such that grippers 99a, 99b, in a tongslike movement, can pickup, hold, and release a wafer from a chuck or robot, etc.

The elevation contrivance 202 encompasses a rectangular base plate to which an actuator 205 centrally is attached to. At the front end corners of the base plate 204, the base plate 204 comprises through holes for attaching the base plate. The actuator 205 is a linear actuator which has a spindle 216 at one side. For to accommodate actuator 205 on the base plate 204, base plate 204 further comprises a centrally positioned through hole, through which the actuator spindle 216 extends. It is the function of the elevation contrivance 202 to elevate manipulator drive 203 in a direction perpendicular to a plan defined by the movement of the arms 207a and 207b. Therefore the elevation contrivance 202 is affixed to the top side of the housing 227 of the manipulator drive 203 via its spindle 216 and linear guides 210 (linear guide 210a to the right side and linear guide 210b to left side of the actuator). The linear guides 210a, 210b consist of a pin 212 (212a and 212b) (Fig. 9) and a guide 211 (211a and 211b) which guide the elevation movement. The pins 212 are symmetrically arranged with respect to the linear actuator 205 and extend through respective wholes, which are comprised by the base plate 204. Pins 212 are attached to base plate by guide clamps 206 (206a and 206b), and co-operate with the guides 211, which are affixed to the top surface of housing 227 by means of

flanges. Thus actuator 205, with its spindle, can move the housing 227 of manipulator drive 203 up and down if the actuator is appropriately charged by current via cable 214. On the front side of housing 227, in central position on the 5 front, a holder 213 is attached to the housing 227, which carries a scanning sensor 226 that detects or checks the presents of wafer 215, i.e. it detects whether the arms 207 bear a wafer or not. The sensor 226 is fastened to the free end of holder 213, whereby the holder extends from housing 10 227 such that at least wafers or disks with two different diameters, for instance of 200mm and 300mm, can be detected.

Turning now to Fig. 9, wherein the back side of the swapping apparatus FSS-2 of Fig. 8 can be seen. Note that 15 there, the back plate of housing 227 of the manipulator drive has been removed. Thus Fig. 9 freely shows the functional elements of manipulator drive 203. Therein two sides can be differentiated. The mechanism on the right drives the right arm 207a and the mechanism on the left drives the left arm 207b. The technical features of both sides are the same. 20 Therefore the following description of the technical parts will be restricted to just the right hand side. The driving mechanism on the right side comprises a motor or linear actuator 217a (217b) which is held by bracket 248a (248b), and which drives a lead screw 218a (218b) that co-operates 25 with the female thread of bracket nut 219a (219b), whereby, depending on the sense of direction of lead screw 218a (218b) bracket nut 219a is either moved to left or to right. Lead screw nut 220a (220b) secures bracket nut 219a (219b). 30 Bracket nut 219a (219b) itself is connected with linear slide 221a (221b) which is joint with guiding member 208a (208b) (see Fig. 8). Linear slide 221a (221b) is guided by linear guide 222a (222b) in parallel to lead screw 218a (218b). Two pairs of limit sensors 223a (223b) and 224a (224b) are 35 arranged along the guiding way of linear slide 221a (221b).

The sensors 223a (223b), 224a (224b) interact with flag sensor 225a (225b) which is lodged to linear slide 221a (221b). The first pair of sensors 223a (223b) is positioned along the guide way such that a 200mm wafer can be either released/picked or held by the arms' grippers 99b. This means that, in case when the bracket nut 219a (219b) and guiding member 208a (208b), respectively, is moved to the left side (the same is valid if guiding member 208b of the left side is moved to the right) the inner sensor 223a' (223b') of the pair 223a (223b) indicates via sensor flag 225a (225b) that arm 207a (207b) or gripper 99a (99b) has the right position to hold wafer 215 (Fig. 8), whereas if the guiding member 208a (208b) or gripper 99a (99b) is moved to the right (left) the outer sensor 223a'' (223b'') of the pair 223a (223b) indicates via flag sensor 225a (225b) that the guiding member 208a (208b) or gripper 99a (99b) has moved far enough that there will be no interference with the wafer left on a chuck or to be picked from a chuck, and thus, in any case, no damage of the wafer 215 will occur. The second pair of sensors 224a (224b) has the same functionality for wafers or disks with a diameter of 300mm. Thus the inventive swapping apparatus advantageously includes the possibility to swap disklike members or wafers of different diameters. Moreover, a further sensor or limit switch 249 at the inside of the top plate of the housing 227 is provided which indicates the distance between the manipulator drive 203 and the elevation contrivance 202.

Another advantageous feature of the FSS-2 according to Figs. 8 and 9 is the fact that the inventive swapping station can be front as well as back loaded. Structurally this is provided by the fact that the arms 207a and 207b are formed and hanged up on the manipulator drive 203 such that the grippers 99a, 99b are free accessible from the front and the back side of housing 227. For that function the part of

guiding members 208a and 208b extending through the guiding apertures of the housing 227 is cranked, and the legs 210a, 210b are attached to the adjustable extension members 209a, 209b, which link the respective guiding members 208a, 208b and legs 210a, 210b, such that an offset of the grippers 99a, 99b and the legs 210a, 210b with respect to the outer side of base plate 251 of housing 227 is achieved. The offset preferably is 10mm.

10 Although not shown in Fig. 8 or Fig. 9, the displayed apparatus of course also comprise appropriate interfaces and cables connected thereto for a remote control of the apparatus' movements.

15 A slightly different embodiment of the invention, if compared to that according to Figs. 8 and 9, is shown in Fig. 10. Therein a schematic top view of the manipulator drive together with the tongslike arms of Fig. 8 is illustrated which differentiates in the fact that not two motors or
20 linear actuators 217a, 217b are used but only one actuator 217 to drive the twin lead screws 218a, 218b and thus the arms 207 or grippers 99 on the right and on the left in Figs 8 or 9. However for to get the tongslike movement of the arms 207a and 207b the lead screws preferably have different
25 thread leads, i.e. the twin lead screws could be for instance to the right 218a left-hand threaded and that to the left 218b right-hand threaded or vice versa. Of course, the same effect could be achieve if the single actuator or motor 217 were able to provide two different driving directions for the
30 two twin lead screws 218a and 218b with the same lead each. Thus, if driven by the motor 217, twin lead screws 218a and 218b cause an anti-parallel tongslike movement of the arms 207a, 207b.

In the embodiment of Fig. 11 the single motor or actuator 217 with two driving shafts 229a and 229b (Fig. 10) for the twin lead screws 218a, 218b is replace by a motor 217 with merely one driving shaft 229 which drives a pulley 230 that by means of a belt drive 231 runs a driven pulley 232. The driven pulley is axially connected with the two twin lead screws 218a and 218b, as it is case with respect to motors according to Fig. 8 to 10. This kind of assembly just provides one sense of rotation of the driven pulley 232 and the twin lead screws 218a and 218b. Therefore the lead screws 218a and 218b have to have opposed thread leads for a tongslike movement of the arms 207a and 207b.

Fig. 12, as Figs 10 and 11, displays, under omission of the elevation contrivance 202 (Figs 8, 9), a schematic top view of a further embodiment of the manipulator drive 203 and the tongslike arms, whereby, as in Figs. 10 and 11 the elevation contrivance, remains the same as in Figs. 8 and 9. Manipulator drive 203 in Fig. 12 comprises a gear drive 233 and a lever apparatus 234 (234a and 234b) for driving the tongslike arms 235 (235a and 235b). The gear drive 233 consists of two gears 233a and 233b which are in mesh. One of the gears 233b is driven by a pulley 238 via pulley 232, which has a rearward position inside the manipulator 203 on the symmetry axis of the swapping. This pulley 238 is driven via belt drive 236 which transfers the motion of a driving pulley 238 that is attached to and driven by an actuator or motor 237.

The lever apparatus 234 of Fig. 12 encompasses two lever structures 234a and 234b. Part of the lever apparatus or structures 234a and 234b are the tongslike arms 235a and 235b. Each of the lever structures 234a and 234b forms a parallelogram of four levers 240, 241, 242 including the portions 239a and 239b of the arms 235a and 235b. In each of

the lever structures 234, for gripping the wafer by means of the grippers 99a and 99b, the arms 235 and thus the portions 239 (239a and 239b) too are brought into a position parallel to a tangent on the edge of the disk or wafer 215. Each of 5 the lever structures 234a and 234b comprises four cylindrical joints which form the edges of the parallelogram. The joints 245 and 246 at the ends of the levers 242a and 242b parallel to the arms' portions 239a and 239b are locally fixed inside the manipulator drive 203, whereby the levers 242 extend 10 parallel to the gear wheels 233, and whereby the joints 246a and 246b at the ends of levers 242a and 242b opposed to the wafer are locally affixed to the hubs 243a and 243b of gear wheels 233a and 233b. Moreover, with respect to the lever apparatus 234a (234b) lever 241a (241b on the left side), 15 which is parallel to lever 240a (240b on the left side), is pinned to pinion or gear 233a (233b on the left side) by means of attachment point 244.

On the basis of the above described structural 20 elements of the embodiment according to Fig. 12 it is provided that if the actuator 237 drives or turns left gear wheel 233b to the left right gear wheel 233b will be turn to the right. Since lever 241a and 241b are fixed to the respective gear wheels 233a and 233b and are jointly fixed at 25 the hubs 243a and 243b, these levers will be also turned to the respective direction, i.e. lever 241a will turn to the right around turning point 243a, and lever 241b will be turned to the left around turning point 243b, whereupon the levers 240a and 240b being parallel to levers 241a and 241b 30 will follow the movement of levers 241a and 241b around the affixed joints 245a and 245b. By, for instance, turning the levers 241b and 240b to the left the distance of their free ends 247b and 248b will be increased with respect to the wafer 215, which results in a movement of the arm 235b to the 35 left. Since levers 241b and 240b rotate the same angle to the

left, there is a parallel offset of arm 235 to the left. The functional description for the left lever structure 235b is also valid for the right lever structure with the exception that the levers act in opposite direction. This is of course necessary for achieving the tongslike movement of arms upon which wafer 215 is supposed to be gripped or released.

Now reference is made to Fig. 13. Fig. 13 shows the accelerated handling process or transfer of a wafer according to the invention from, for instance, a wafer container-cassette to, for example, an inspection station (both not shown). In that process a wafer which has just been inspected is brought by a chuck of a x-y-stage to an exchange-region EXR. Therein, one of the inventive swapping apparatus FSS-1 or FSS-2 is located and awaits the coming of the wafer 315 (Fig. 13A). The arms 307a and 307b of the swapping apparatus FSS-2 are in their down and open position. Functionally, this could be for instance done be the elevation contrivance 202 and by the manipulator drive 203 according to one of the described embodiments. In Fig. 13B the x-y stage with chuck 320 moves the wafer 315 into the inventive swapping station FSS-2, i.e. between the arms of FSS-2. Swapping station FSS-2 grips and lifts, i.e. front loads the wafer from the chuck by means of the inventive driving mechanism 301 and the arms 307a and 307b. Having just picked the wafer from chuck 320 a new wafer 315' to be inspected is brought to the exchange-region EXR by robot 350. The robot places new wafer 315' onto chuck 320 beneath swapping station FSS-2 (Fig. 13C). Now the x-y-stage retracks chuck 320 from the exchange position EXR to the inspection station with new wafer 315' (Fig. 13D). After or parallel to this robot 350 picks the old wafer 315 from the FSS-2 and retracks its arm with the old wafer 315 from the inventive swapping station FSS-2 (Fig. 13E). In the following, chuck 320 and robot 350 are both retracked, whereby robot 350 holds old wafer 315 and the chuck the new

wafer 315' (Fig. 13F).

Figs 14A to 14F show the same process as Figs 13A to 13F with the difference that the swapping station FSS-2 or 5 FSS-1 is first back-loaded with a new wafer 315', and afterwards releases the new wafer 315' to the chuck. Thus, the first picture Fig. 14A shows robot 350 coming to the exchange region EXR, where the FSS-2 waits to accept wafer 315'. Having arrived at the exchange position the robot, 10 beneath the FSS-2, places the new wafer on the FSS-2, which is, with its arms, in the up and hold position (Fig. 14B). Parallel and/or after this the x-y-stage with its chuck comes to the exchange region EXR or exchange position, where the arms of the FSS are positioned. The robot handler 350 15 retracks from the swapping station FSS-2 (Fig. 14C). Arrived at the exchange position, the robot gets the old wafer 315 from the chuck of the x-y-stage (Fig. 14D). Now, Robot 350 retracks its arm with the old wafer 315 from the chuck 320, and the FSS places new wafer 315' onto the chuck 320 of the 20 x-y-stage (Fig. 14E). Then, x-y-stage retracks chuck 320 with new wafer 315', and Robot 350 brings the old wafer 315 to, for instance, a container-cassette.

Although the preferred embodiments of the invention 25 has been described, it is to be understood that the invention is capable of other adaptations and modifications within the scope of the appended claims.

CLAIMS.

1. Apparatus for swapping at least one disklike member, e.g. a wafer, said apparatus comprising at least two tongslike arms for accepting and holding the disklike member and a driving-mechanism adapted to drive said arms, wherein said mechanism is adapted to provide a first and a second movement of said arms, whereby said first movement comprises a vertically oriented movement of said arms from an up to a down position and vice versa, and whereby said second movement comprises a horizontally oriented tongslike movement of said arms.
2. Apparatus according to claim 1, wherein said driving mechanism comprises a lever apparatus and/or spindle means for controlling said movement of said arms.
3. Apparatus according to claims 1 or 2, wherein said driving mechanism, when performing its horizontally oriented tongslike movement, is adapted to move said arms into at least one hold and into at least one release position and vice versa.
4. Apparatus according to one of the preceding claims, wherein said driving mechanism is adapted to move said arms into two different hold positions and into two release positions corresponding to said hold positions.
5. Apparatus according to one of the preceding claims, wherein said driving mechanism comprises an elevation contrivance and a manipulator drive.
6. Apparatus according to one of the preceding claims, wherein said arms comprise extension members.

7. Apparatus according to one of the preceding claims, wherein said arms comprise gripping means for gripping said disklike member.

5 8. Apparatus according to claim 7, wherein said gripping means comprises at least one grooved circular ring section adapted to the radius or diameter or circumference of said disklike member to be gripped.

10 9. Apparatus according to one of the preceding claims, wherein said apparatus comprises a housing including at least a part of the driving mechanism.

15 10. Apparatus according to one of the preceding claims, wherein said arms and/or gripping means comprise a tongslike structure which enables the arms to be front- or to be back-loaded

20 11. Apparatus according to one of the preceding claims, wherein said driving mechanism comprises at least one driving motor.

12. Apparatus according to one of the preceding claims, wherein said arms are affixed to said driving mechanism

25 13. Apparatus according to one of the preceding claims, wherein the apparatus comprises sensor means for detecting said disklike member and/or for detecting the position of said arms.

30 14. Apparatus according to one of the preceding claims, wherein the apparatus comprises or is connectable to control means for controlling said movement of said arms.

15. Method for handling or transporting disklike members, e.g. wafers, comprising the following steps:

- a) Positioning an apparatus according to one of the preceding claims into an exchange-region for exchanging dislike-members,
- b) transporting a first disklike member with first transport means from a first position to the exchange region,
- c) loading said first disklike member into said apparatus
- d) transporting a second disklike member with second transport means to said exchange region,
- e) loading said second disklike member from said second transport means to said first transport means,
- f) transferring said first disklike member from said apparatus to said second transport means,
- g) transporting said second disklike member to a second position with said second transport means.

16. Handling line for carrying out a method according to claim 15 comprising first and second transport means, and an apparatus according to one of claims 1 to 14.

17. Handling line according to claim 16 wherein said first transport means or said second transport means is either a x-y-stage or chuck or a robot.

ABSTRACT.

The invention is an apparatus for accepting and transferring at least one disklike member. The apparatus 5 comprises a pick- and place-mechanism including gripping means. In effect the mechanism is adapted to provide pick- and/or place-cycles, which during operation provides a movement of said gripping means between an up and a down position and vice versa, whereby, in the down position, said 10 gripping means either picks the disklike member from a load position or places the disklike member onto the load position.

Fig. 1

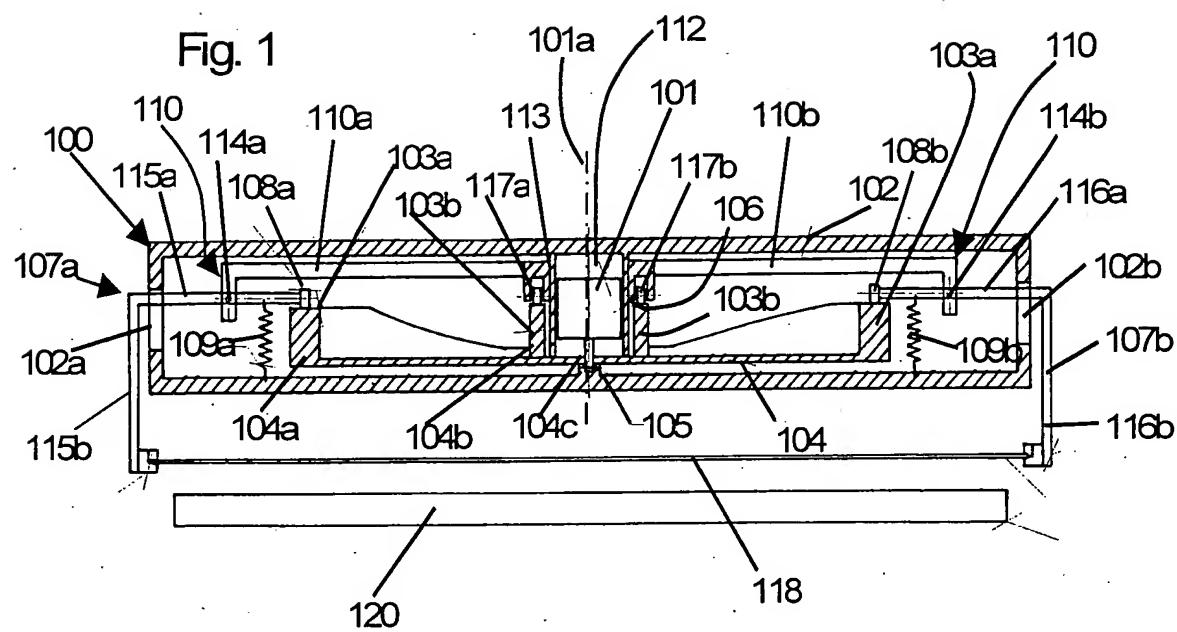


Fig. 2

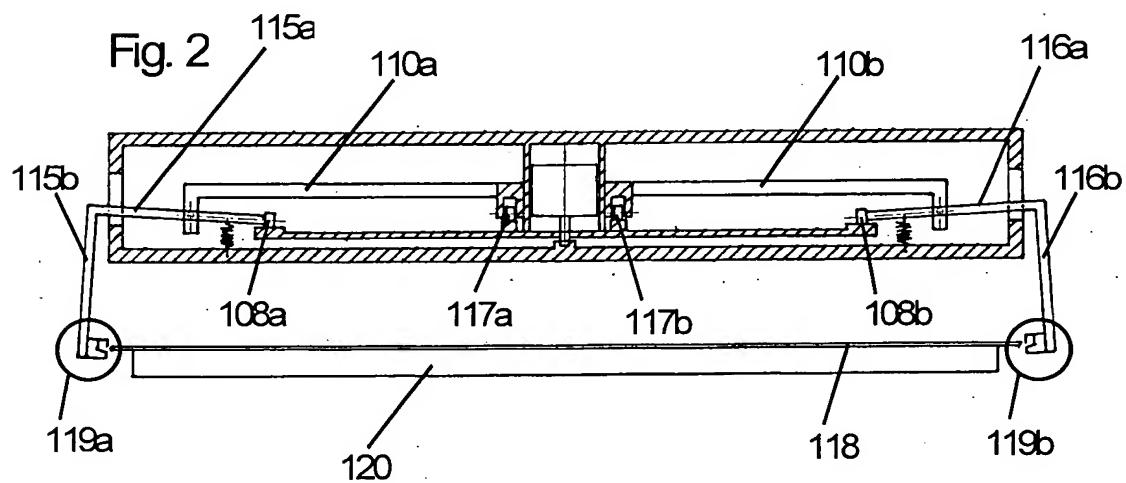


Fig. 3

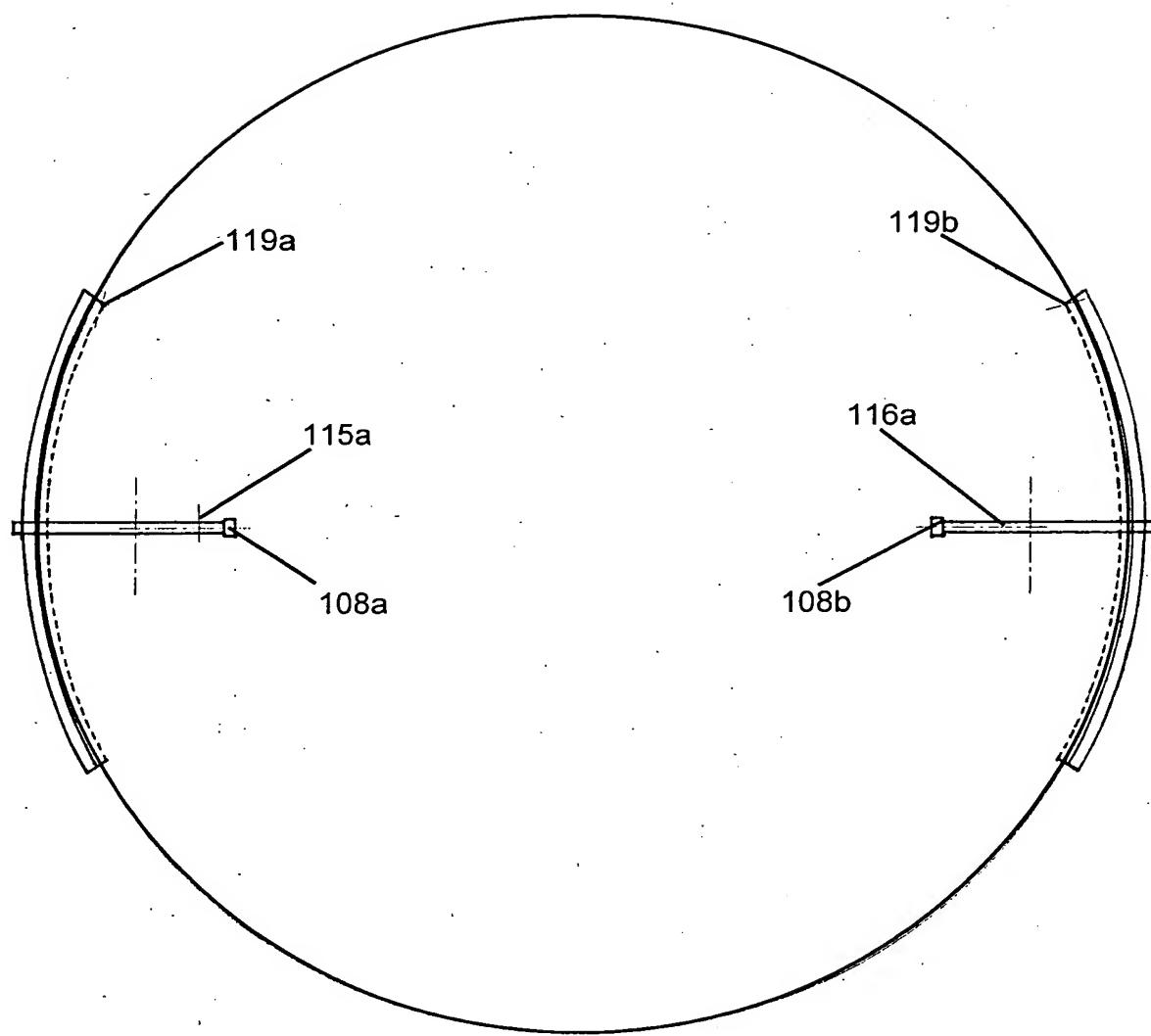


Fig. 4

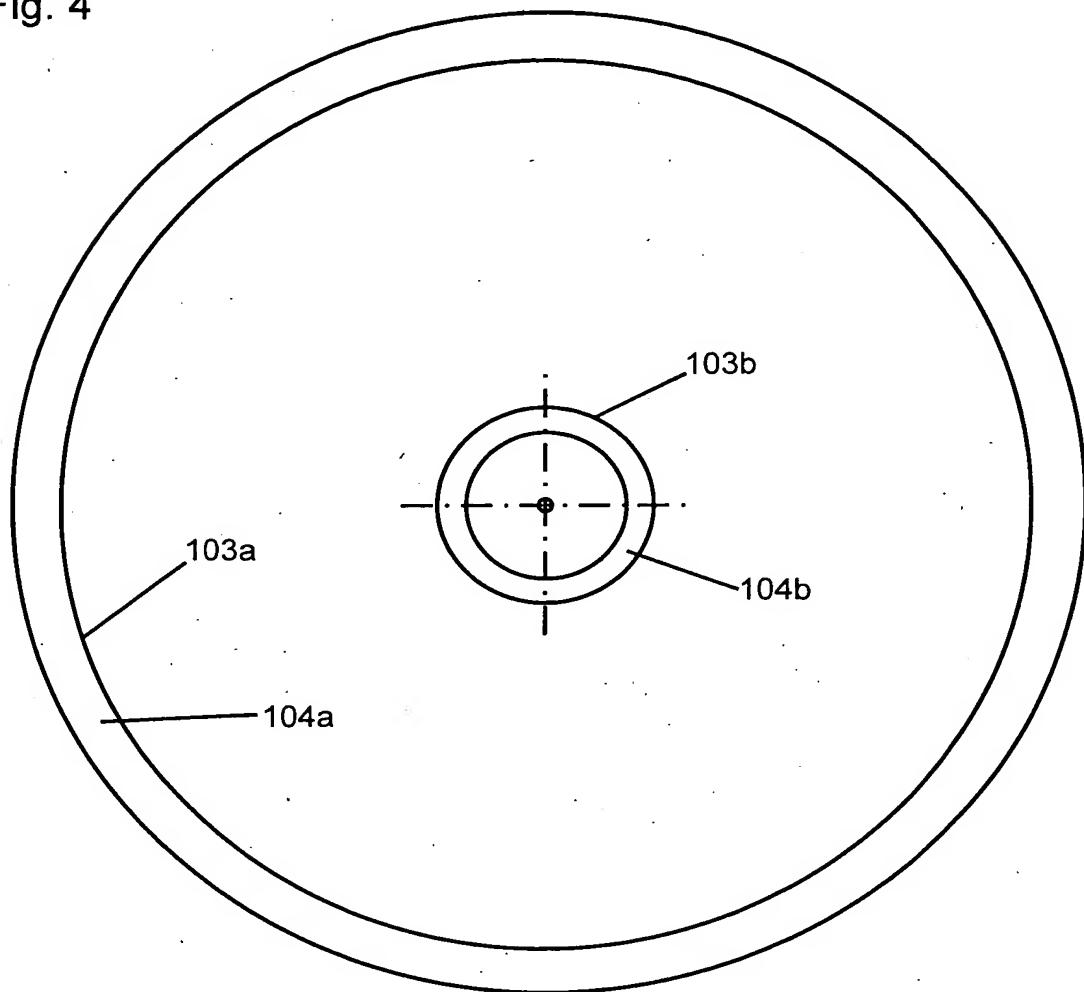


Fig. 5

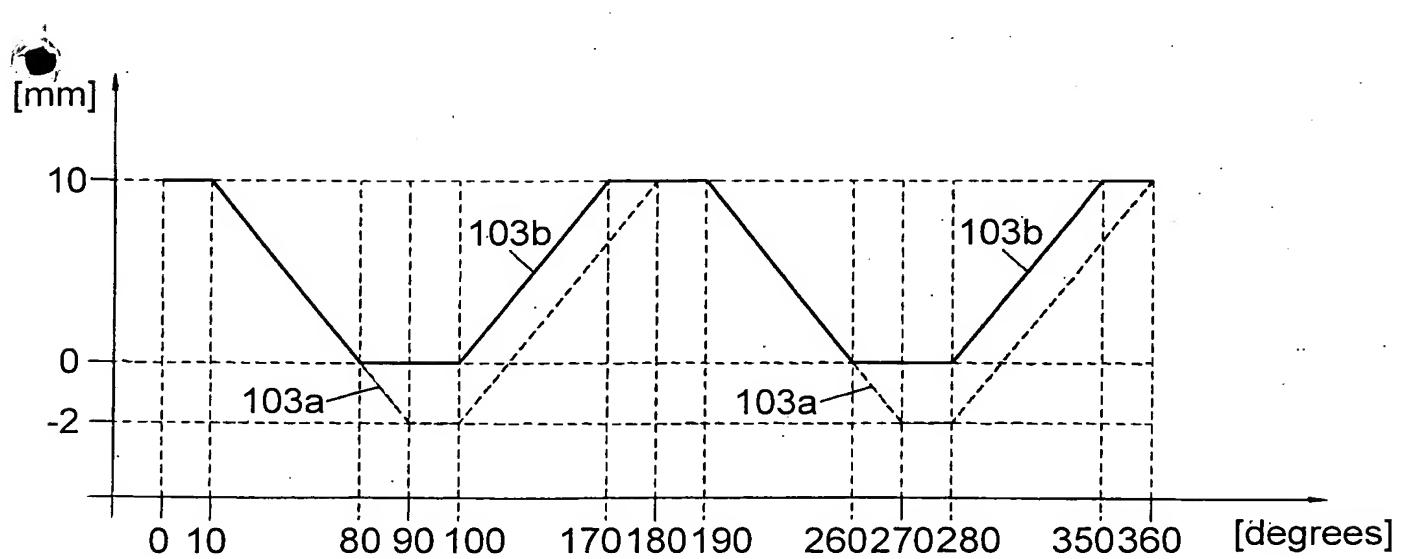


Fig. 6

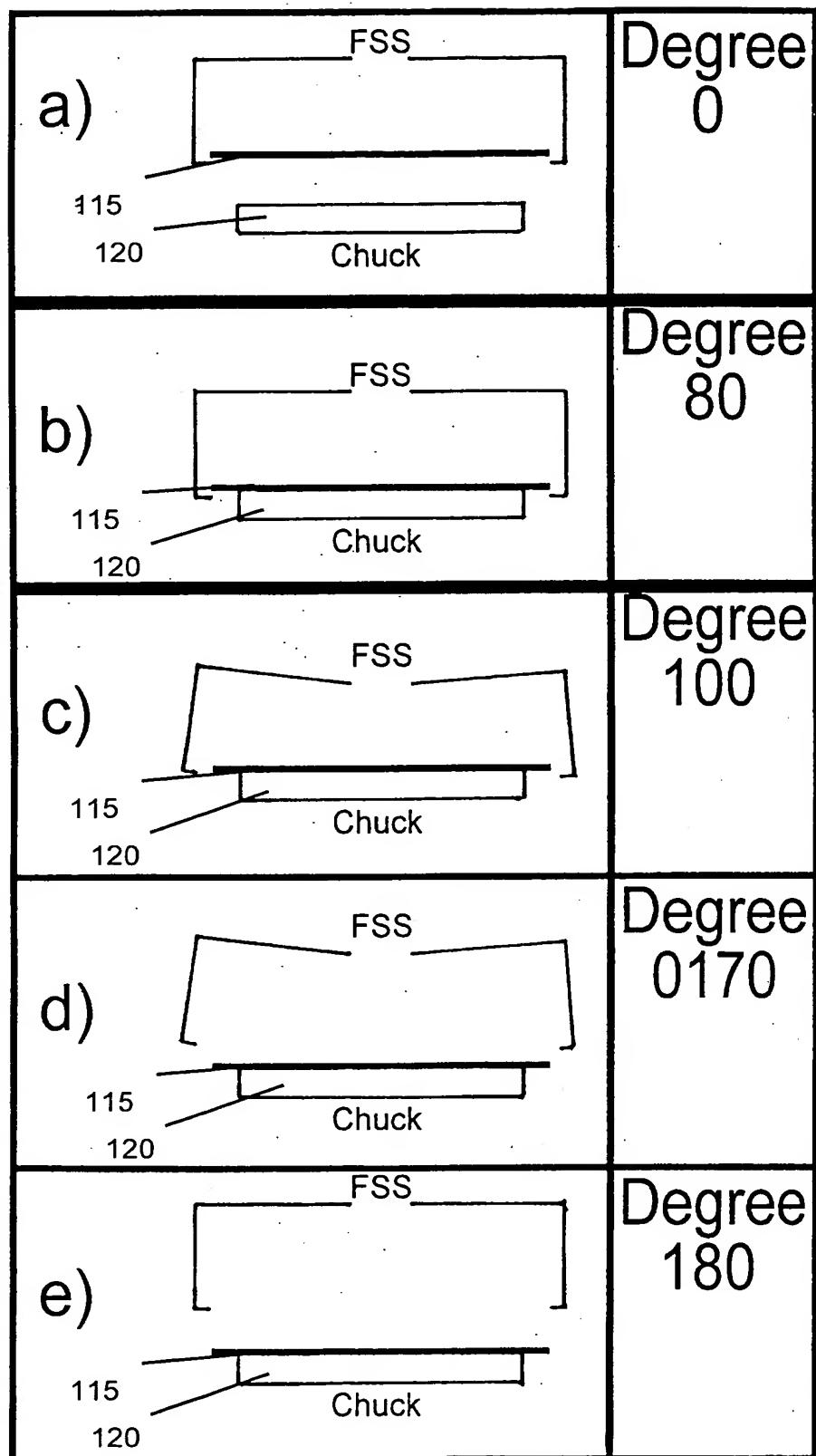


Fig. 7

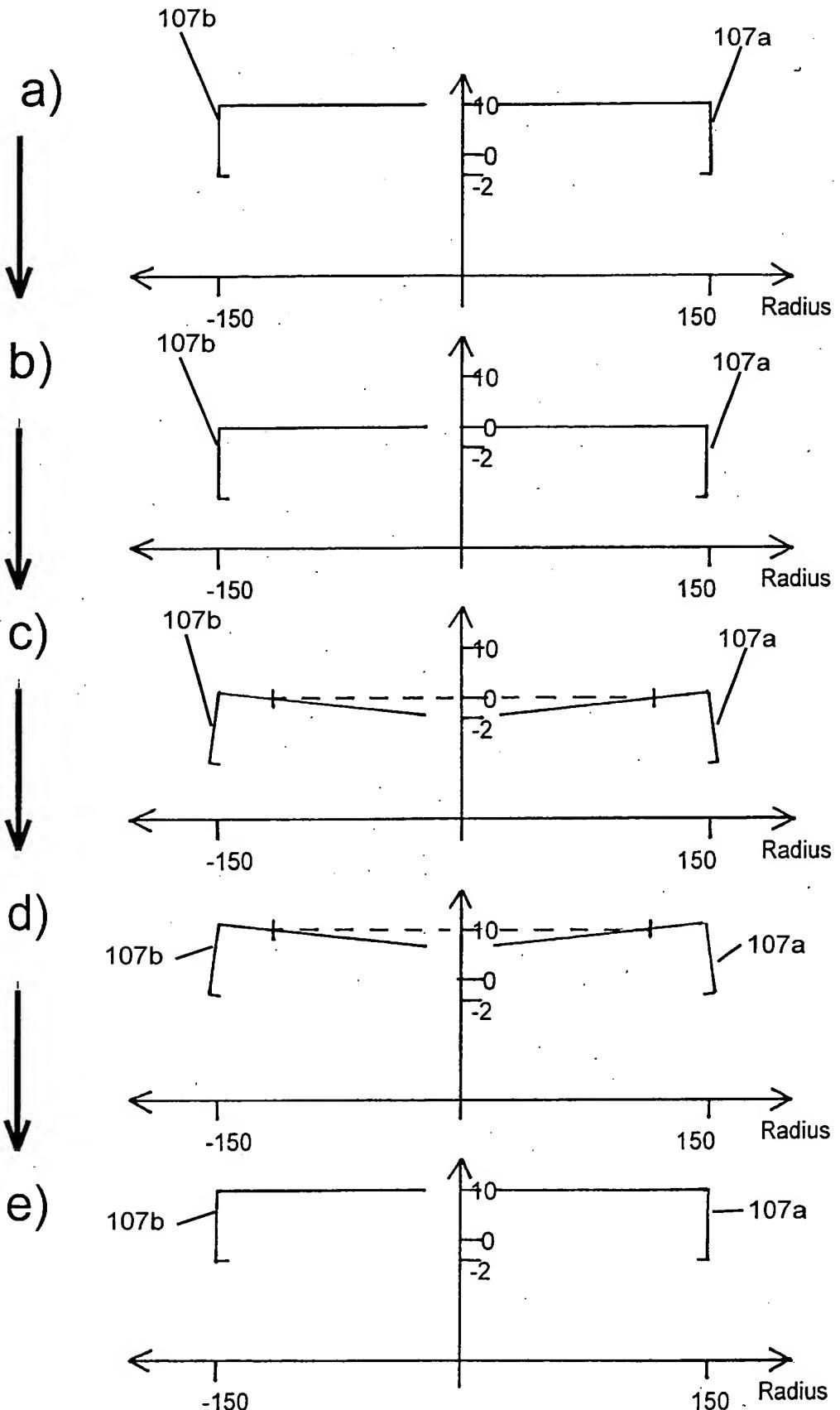


Fig. 8

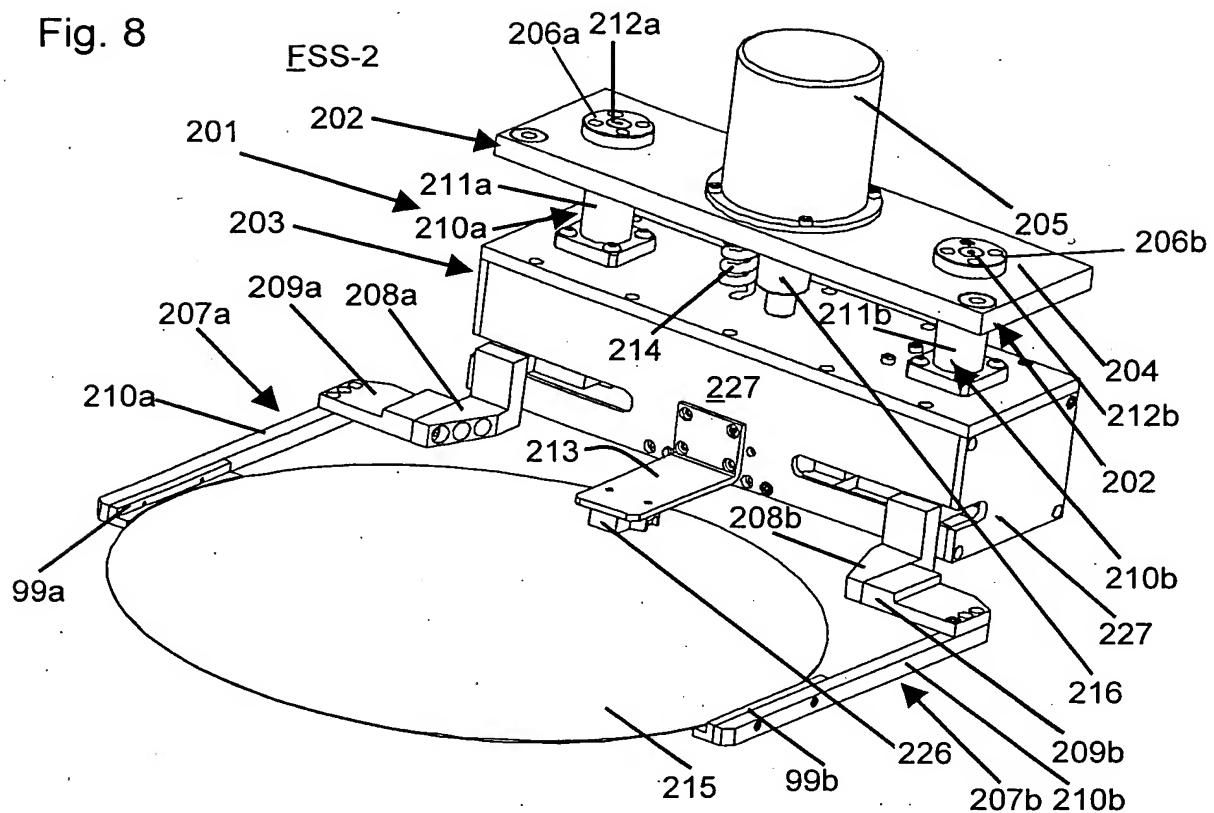


Fig. 9

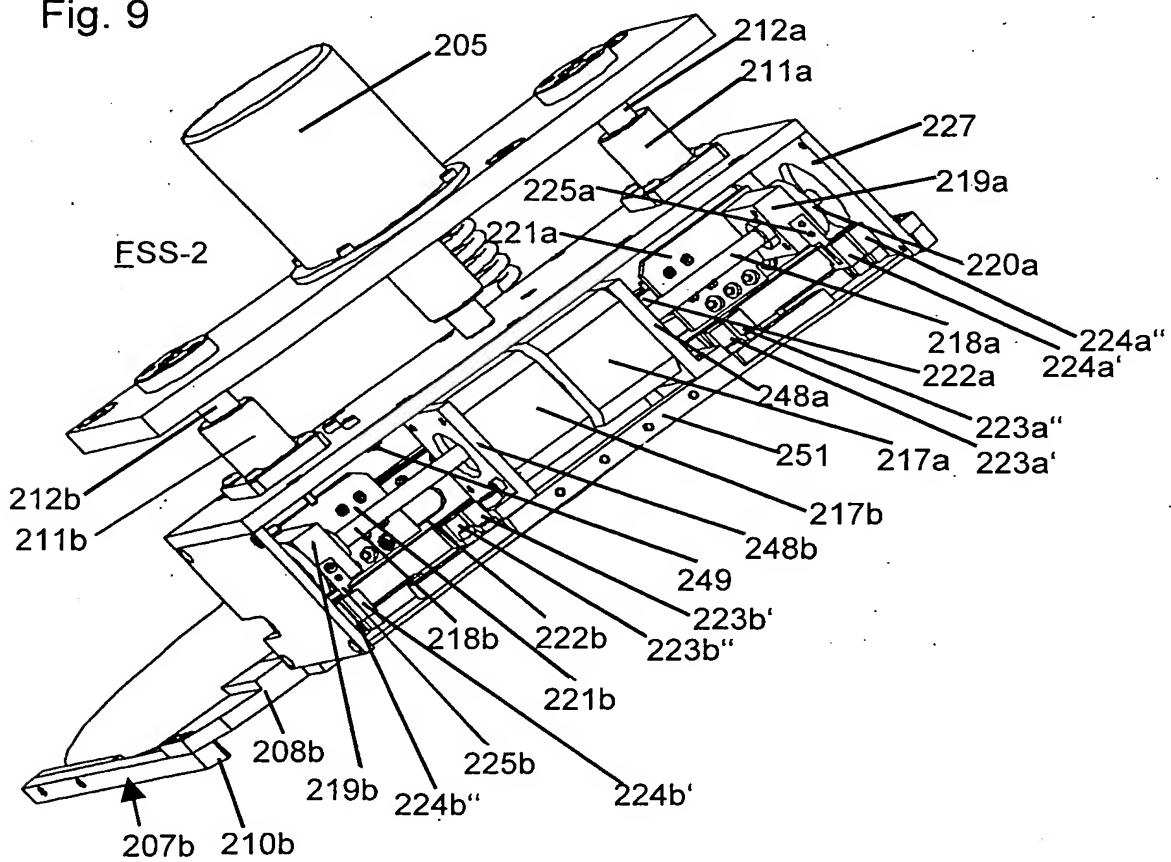


Fig. 10

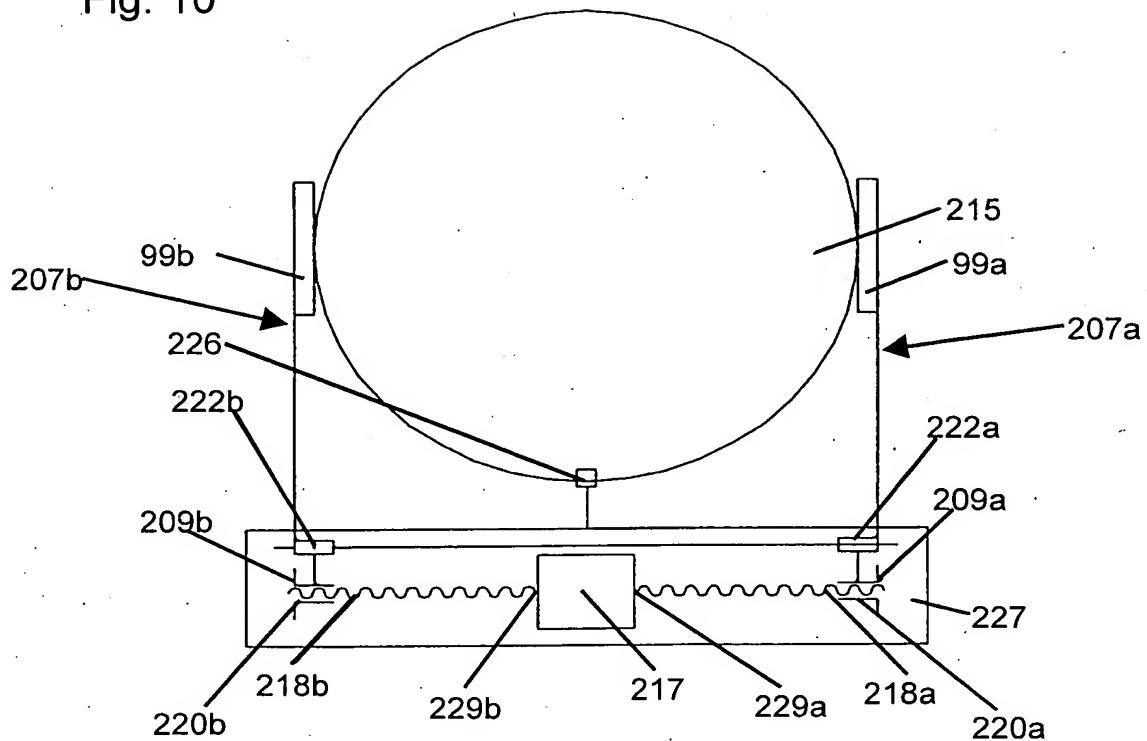


Fig. 11

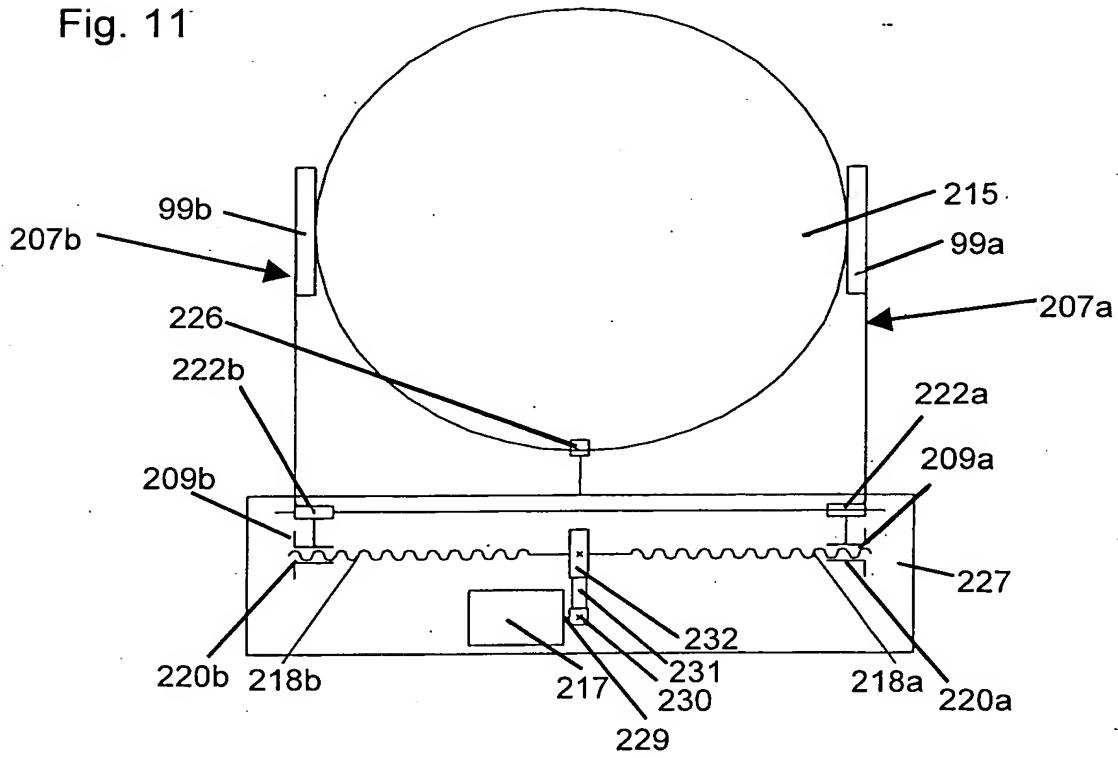


Fig. 12

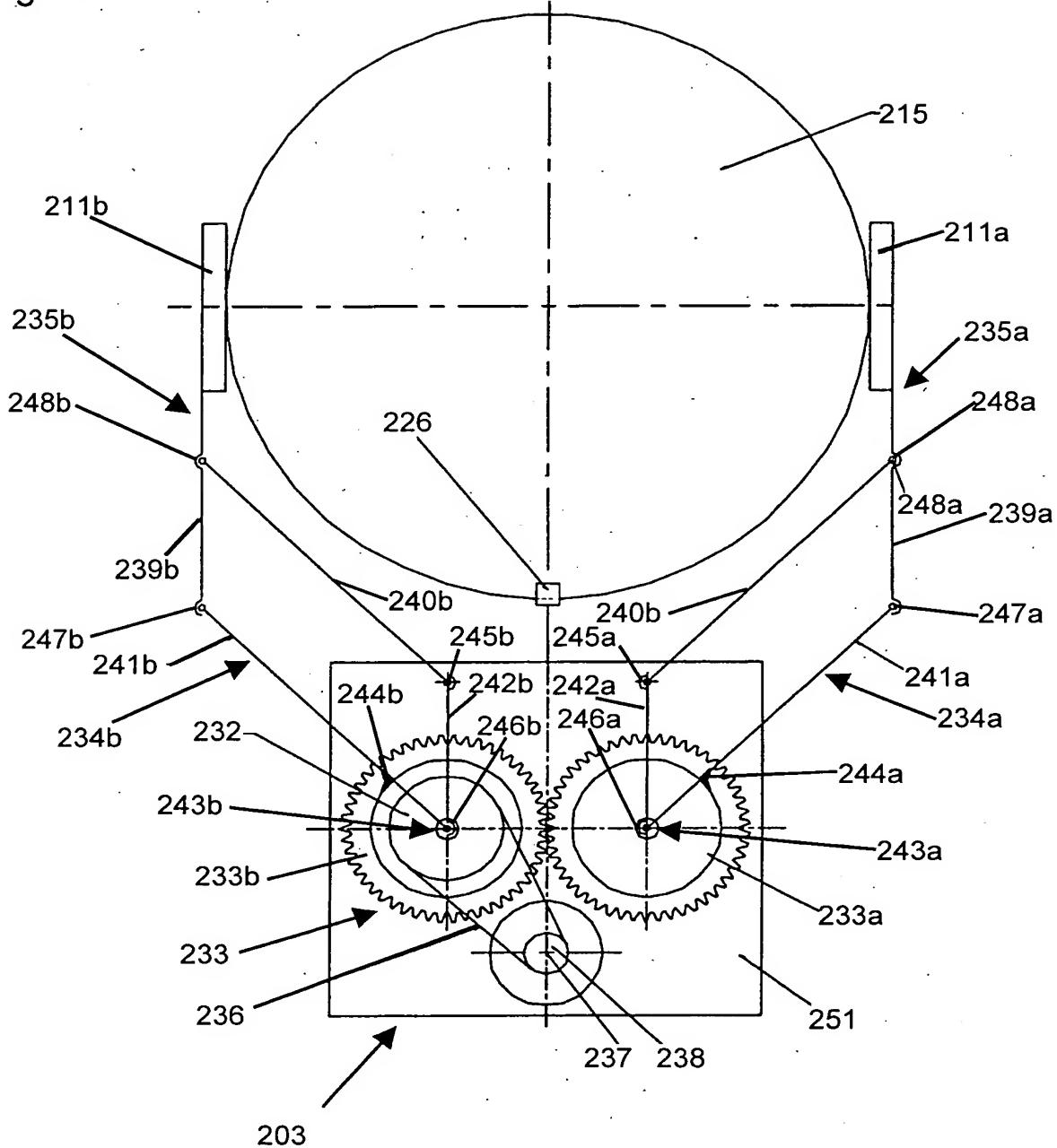


Fig. 13

Fig. 13A

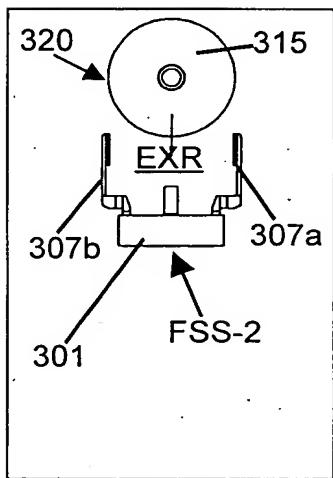


Fig. 13B

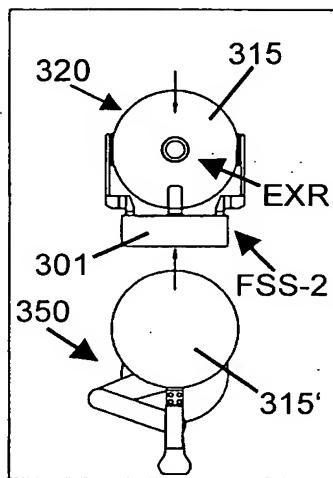


Fig. 13C

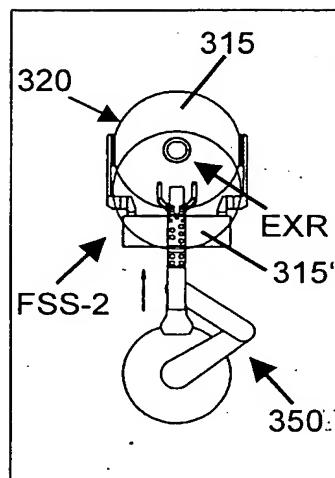


Fig. 13D

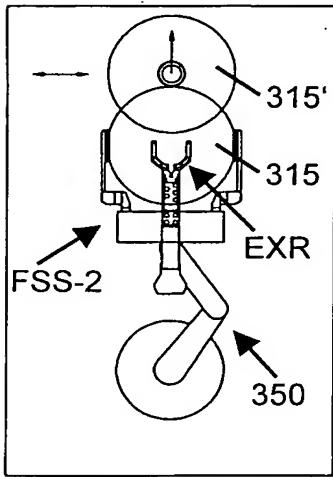


Fig. 13E

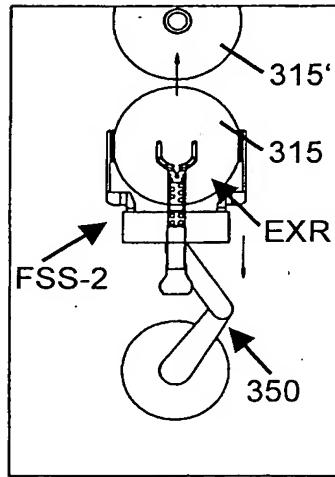


Fig. 13F

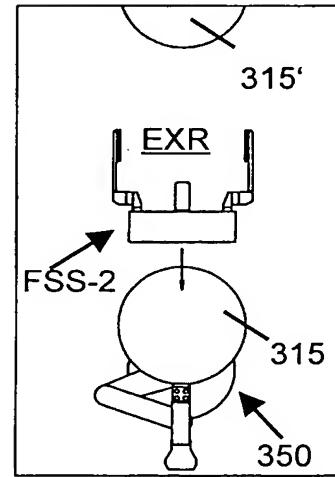


Fig. 14

Fig. 14A

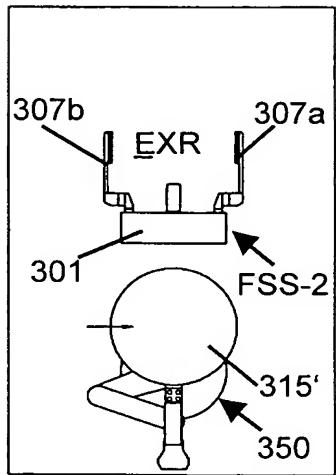


Fig. 14B

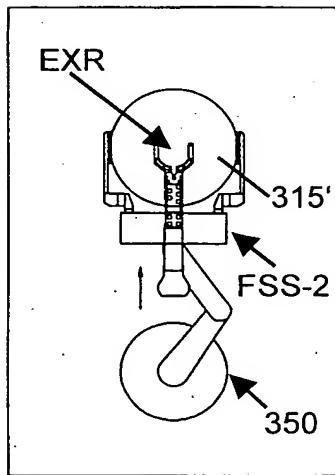


Fig. 14C

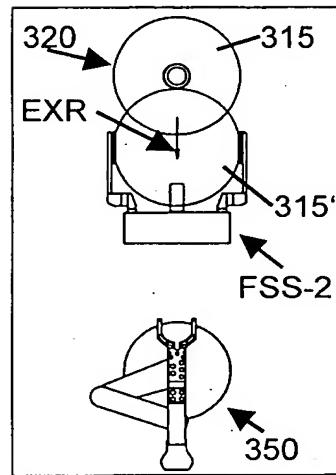


Fig. 14D

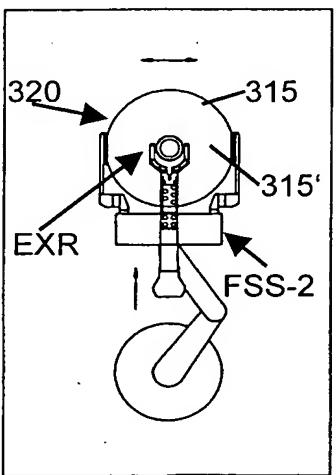


Fig. 14E

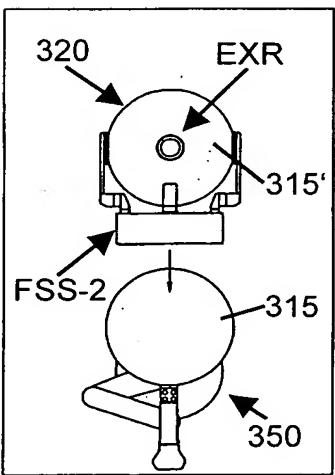


Fig. 14F

